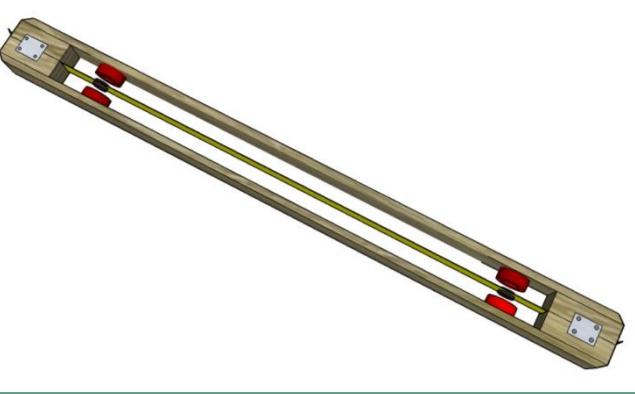




Limit Cycle Oscillations of a Pre-Tensed Membrane Strip

Msc Research Study by Ariel Drachinsky
Under the Guidance of Prof. Daniella Raveh



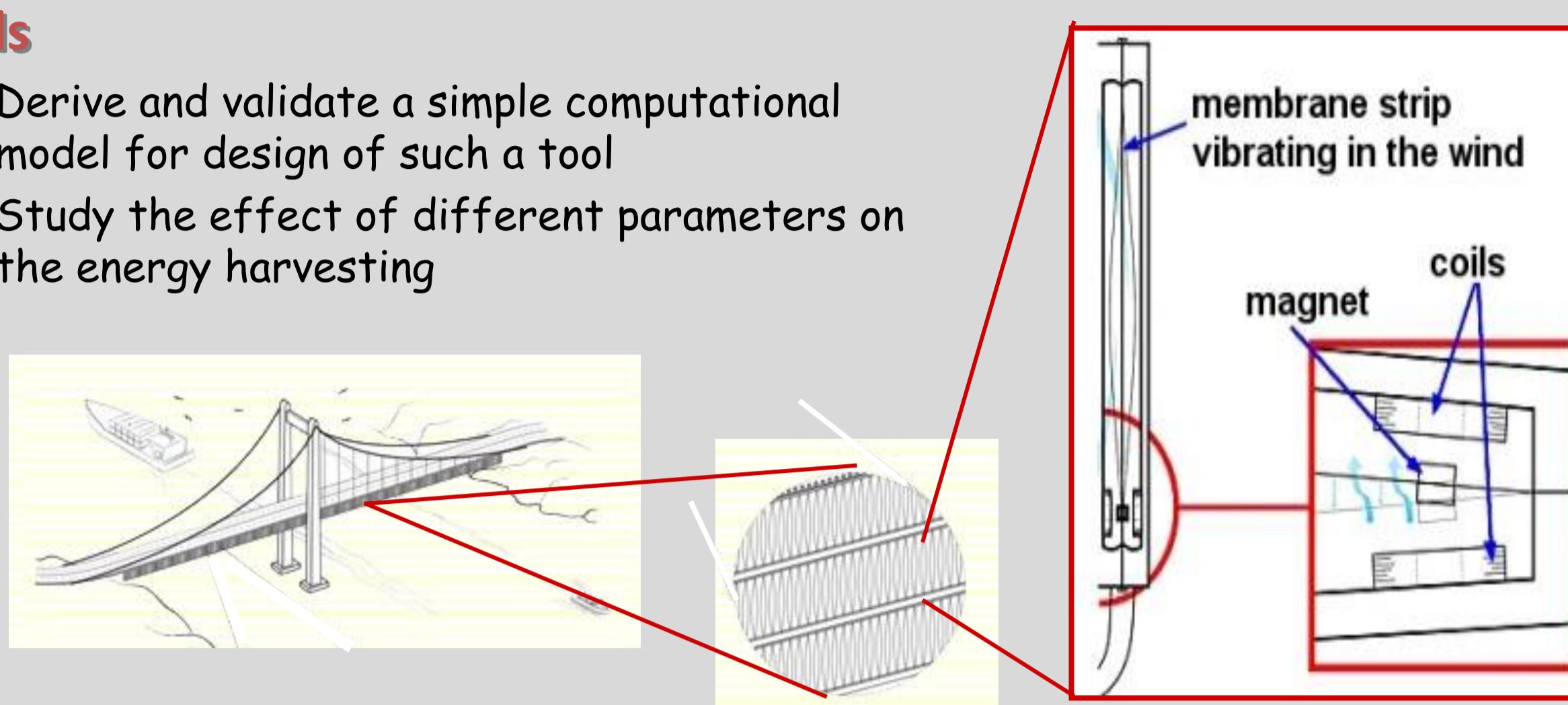
Introduction

Motivation

- Energy harvesting from flutter (LCO)
- Energy harvesting at low airspeed (<10m/s)
- Compact, cheap and green energy generator

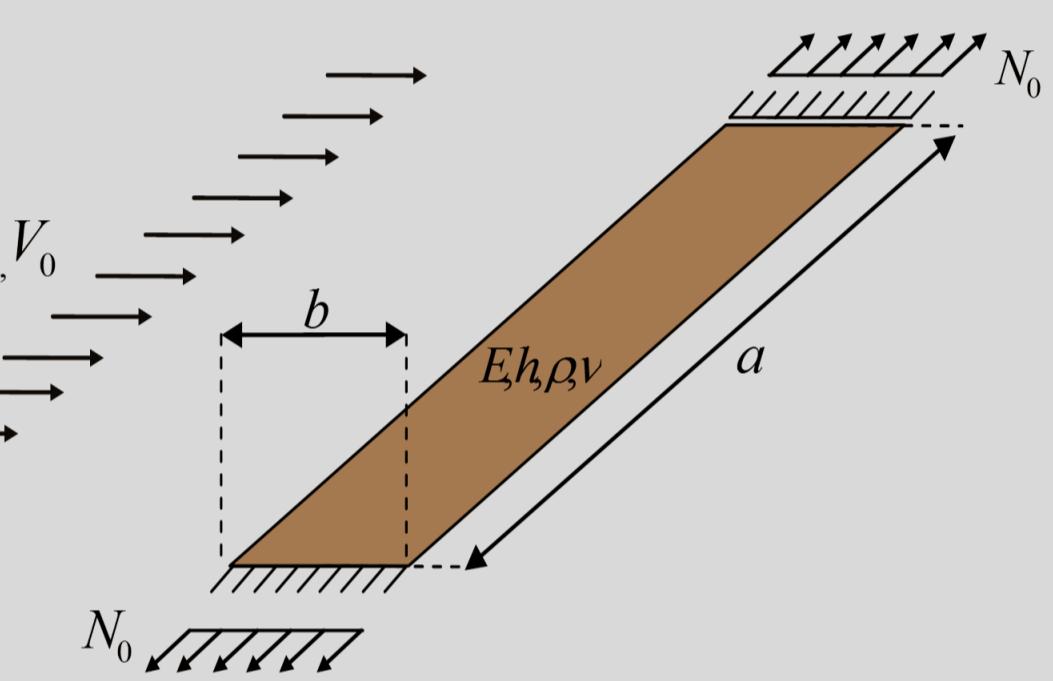
Goals

- Derive and validate a simple computational model for design of such a tool
- Study the effect of different parameters on the energy harvesting



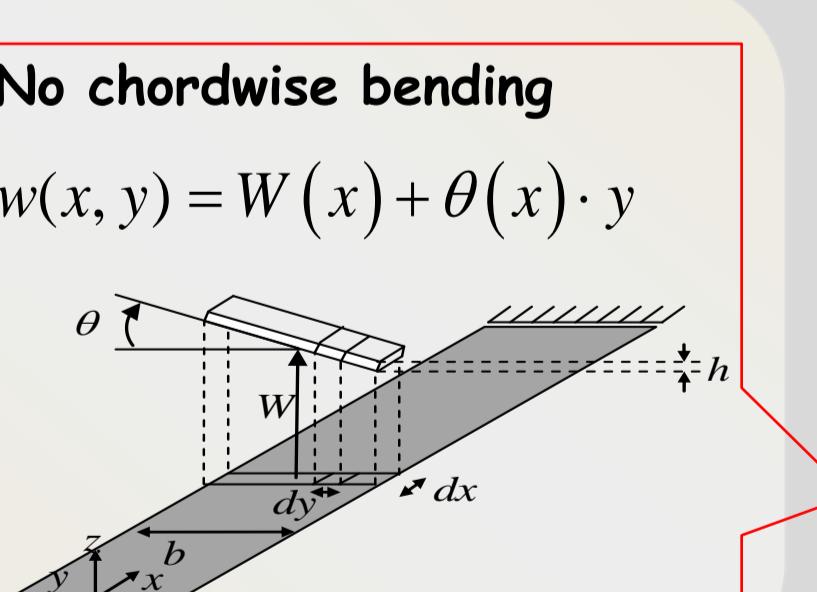
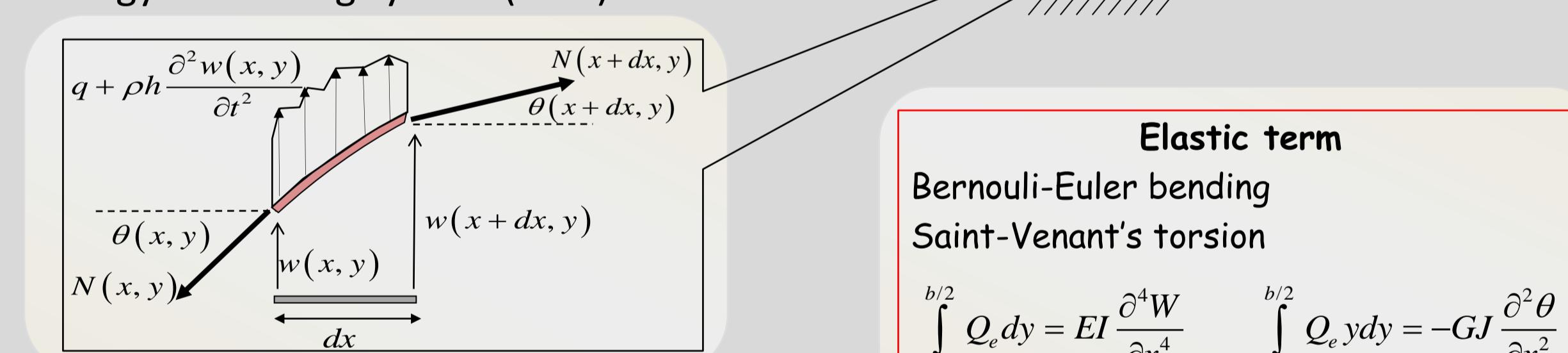
Problem Definition

- A thin membrane of high AR is clamped at the short edges.
- The membrane is pretensed in the long (span) direction.
- Air flows in the direction of the short edge (chord).
- At a certain velocity, the membrane flutters.
- Due to non-linear stiffening effects the oscillations converge to a limit cycle.



Mathematical Model "Beam-String Model"

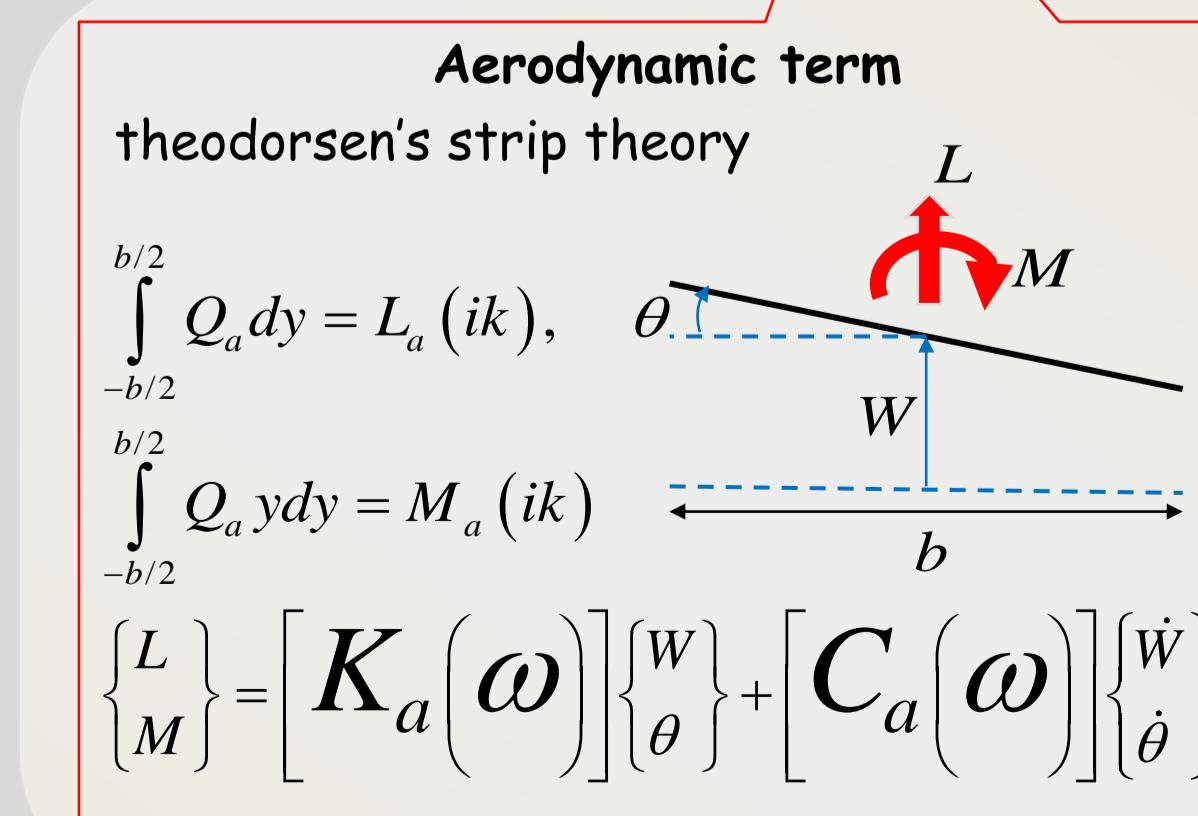
- Linear, elastic material properties
- Chordwise bending is negligible
- Large deformations, small angles small strains
- Potential flow
- Oscillations reach steady state
- Energy harvesting system (coils) is not considered



- Nonlinear geometric stiffness term
- Stiffness due to tension
- Pretension
- Additional tension due to large deformation

$$N(x, y) = N_0 + \Delta N = N_0 + Eh \frac{L-a}{a}$$

$$\approx N_0 + Eh \int_0^a \left(\frac{\partial w(x, y)}{\partial x} \right)^2 dx$$



Contact Information

- arikdra@gmail.com
- Phone: 054-272516

Solution Via Galerkin's Method

- Galerkin's method transforms a PDE to a set of ODE's

$$w = W + \theta \cdot y$$

$$W(x, t) \approx X_1(t)W_1(x) + X_2(t)W_2(x) + X_3(t)W_3(x) + X_4(t)W_4(x)$$

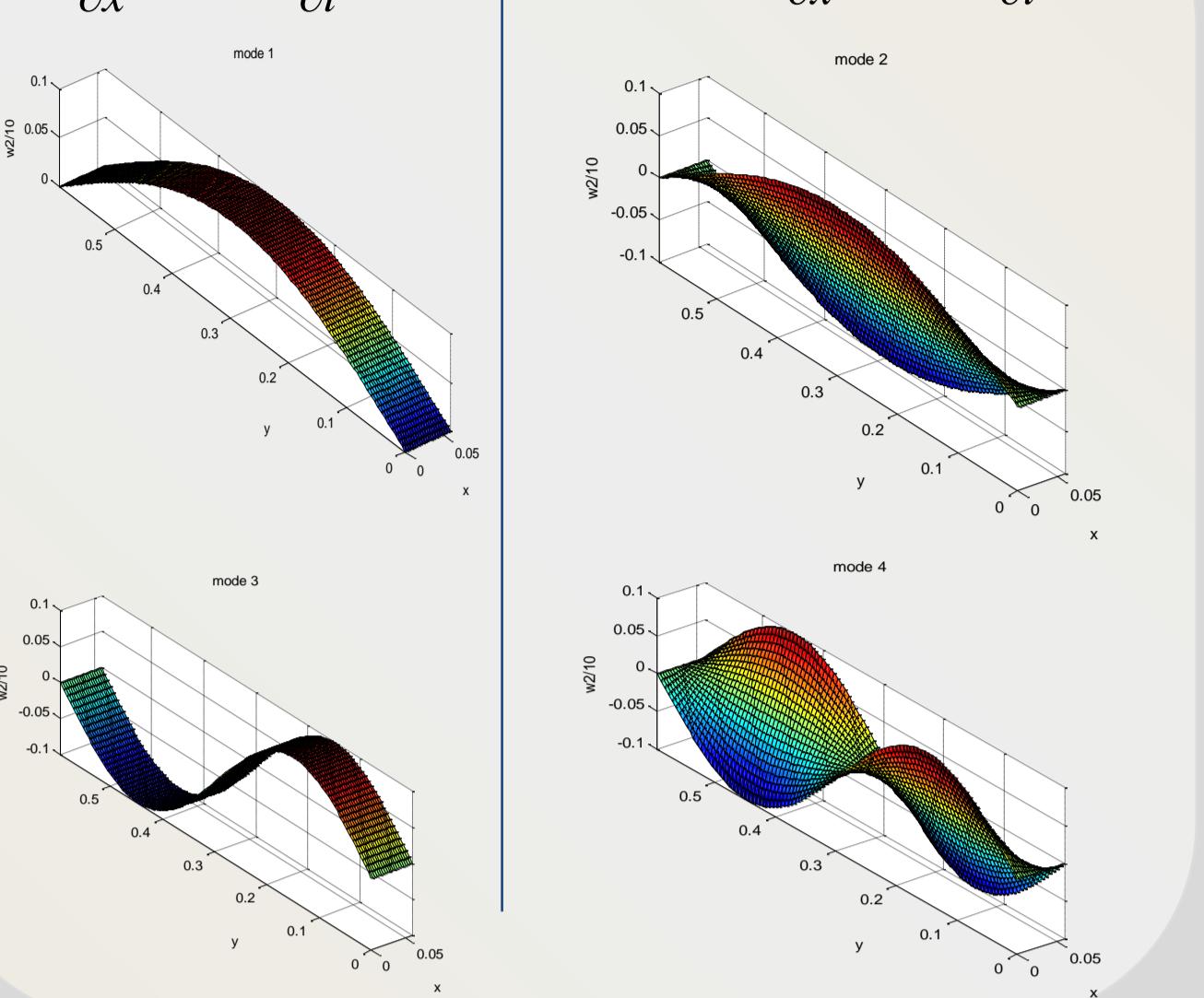
$$\theta(x, t) \approx X_1(t)\theta_1(x) + X_2(t)\theta_2(x) + X_3(t)\theta_3(x) + X_4(t)\theta_4(x)$$

$$\int_0^{b/2} \left[\frac{\partial}{\partial x} \left(N(x, y) \frac{\partial w(x, y)}{\partial x} \right) - \rho h \frac{\partial^2 w(x, y)}{\partial t^2} - Q_e - Q_a \right] w_i dy dx = 0$$

$$\{ \ddot{X} \} + \{ [C] - v[C_s] \} \{ \dot{X} \} + \{ [K] - v^2[K_a] \} \{ X \} + \{ f(\xi X^3) \} = 0$$

$$\sigma_0 A \frac{\partial^2 W}{\partial x^2} - \rho A \frac{\partial^2 W}{\partial t^2} = 0;$$

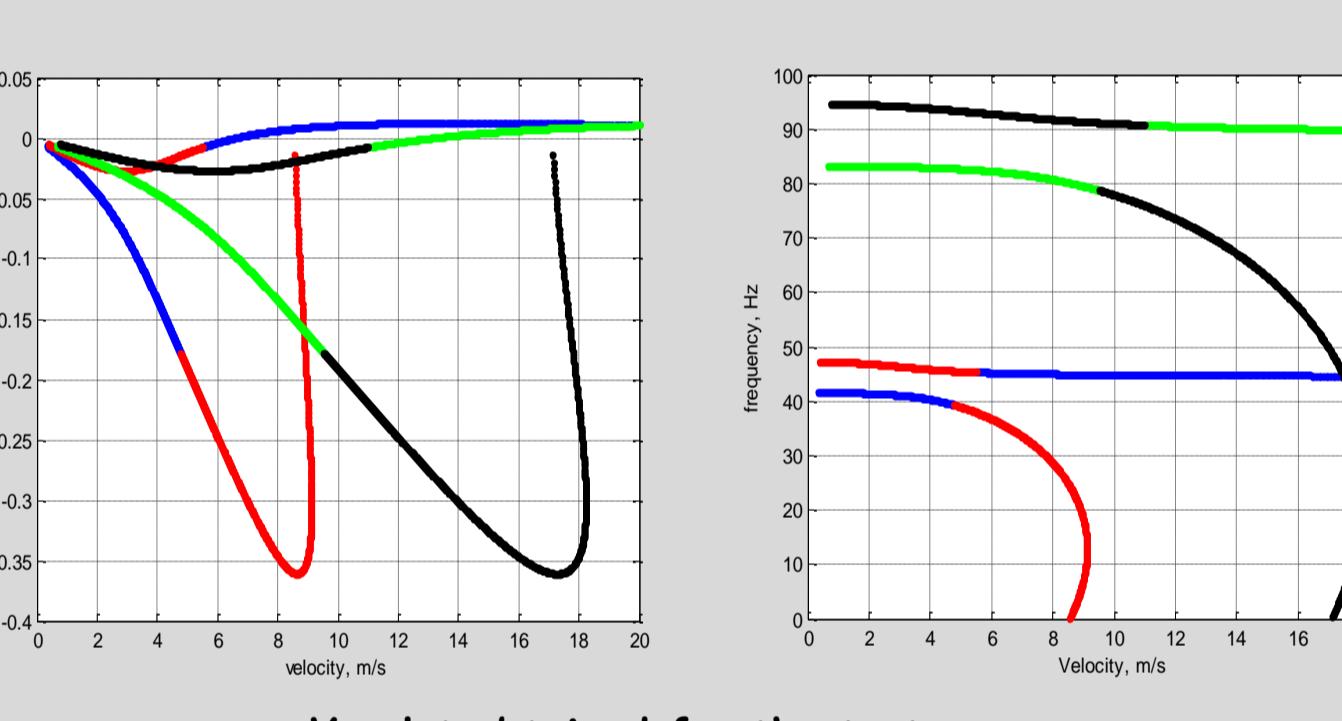
$$(\sigma_0 I_p + GJ) \frac{\partial^2 \theta}{\partial x^2} - \rho I_p \frac{\partial^2 \theta}{\partial t^2} = 0$$



- Simulation time ~3000 periods. (small damping at near-flutter velocities)
- Time step ~0.05 period.
- Aerodynamic terms depend on frequency. -> Solution is followed by a spectral analysis.
- Aerodynamic terms are updated, and the problem is simulated again until convergence.

Test-case

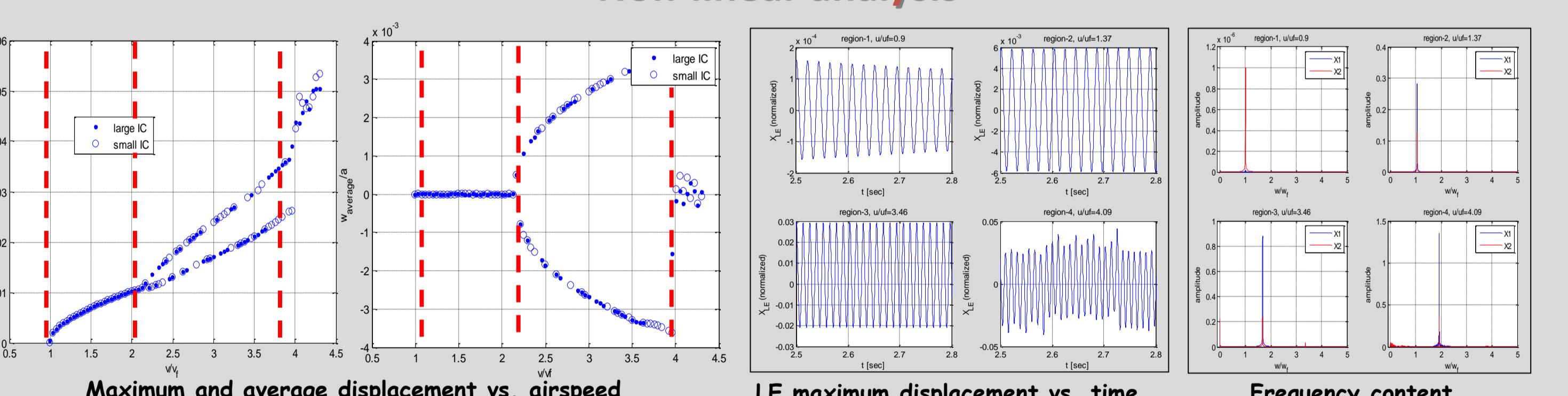
Linear analysis, comparison between 2d and 3d models



Property	a	b	h	α_0	E	p	ζ	ρ_s	v
value	596	25	0.25	3.89	6098	1430	0.0	1.225	0.39
units	mm	mm	mm	MPa	MPa	Kg/m ³	%	Kg/m ³	--

- ✓ Strip theory is adequate
- ✓ Beam model is adequate
- ✓ Bending stiffness is negligible
- ✓ Four modes are sufficient

Non-linear analysis



region	v/vf	Stable. All oscillations decay.
1	0-1	LCO 1st and 2nd modes.
2	1-2.2	LCO 1st and 2nd modes with a static offset (the oscillation is about non-zero deflection). Not obtained in experiment
3	2.2-3.7	Gradual loss of periodicity, turning into chaotic oscillations.
4	>3.7	

Parametric study, effect of preload on power

Power calculation

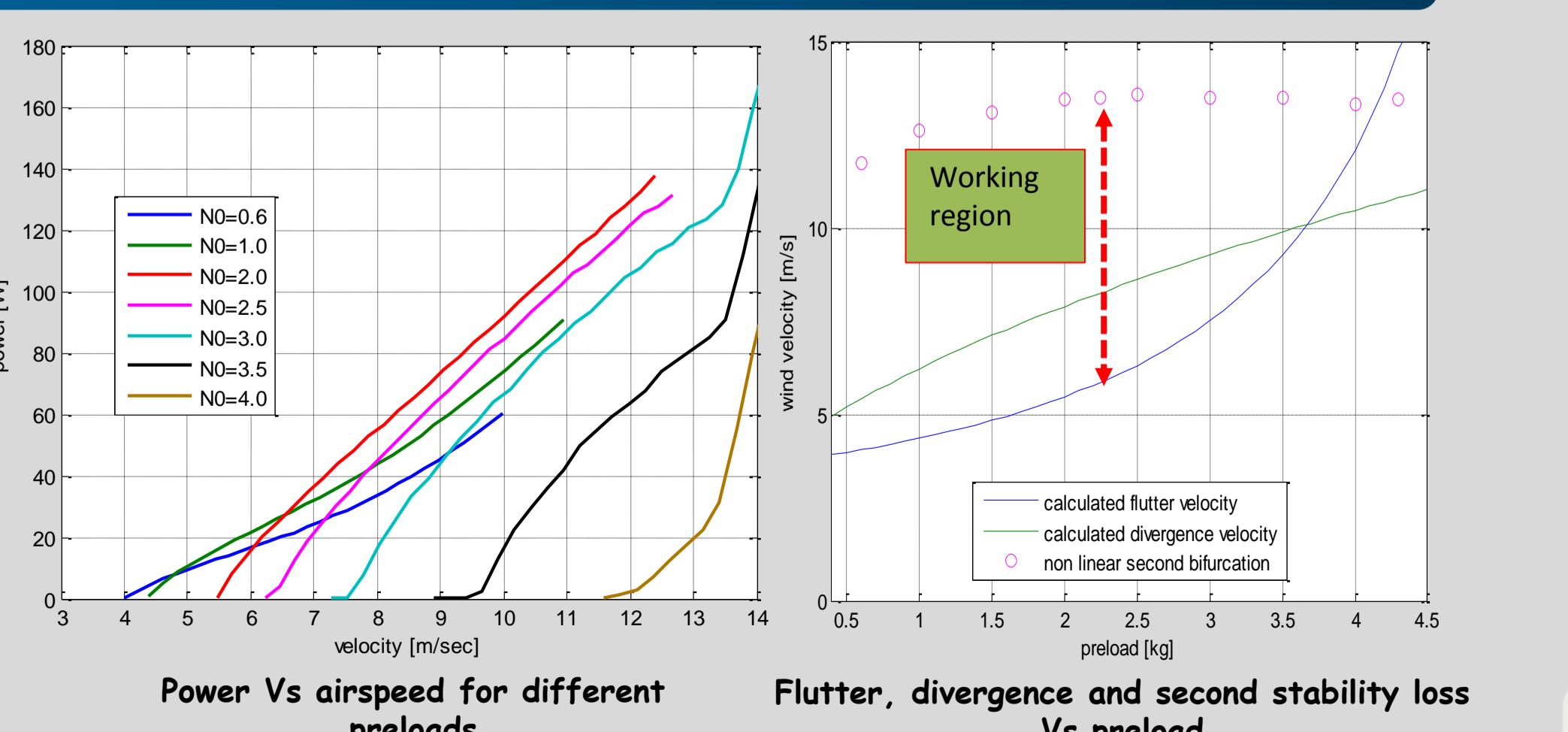
$$\bar{W} = \frac{1}{T} \sum_{i=1}^4 \int_t^{t+T} \left(F X_i (X_1, X_2, X_3, X_4) \cdot \dot{X}_i + F \dot{X}_i (\dot{X}_1, \dot{X}_2, \dot{X}_3, \dot{X}_4) \cdot \dot{X}_i \right) dt$$

$$\int_{-b/2}^{b/2} Q_a dy = L_a (ik), \quad \theta = \frac{M}{Eh}$$

$$\int_{-b/2}^{b/2} Q_a y dy = M_a (ik)$$

$$\begin{bmatrix} L \\ M \end{bmatrix} = \begin{bmatrix} K_a(\omega) \\ C_a(\omega) \end{bmatrix} \begin{bmatrix} w \\ \dot{w} \end{bmatrix}$$

Top boundary for the power harvesting.



- Optimal preload is dependent on designed velocity region
- Higher preloads yield smaller working region.

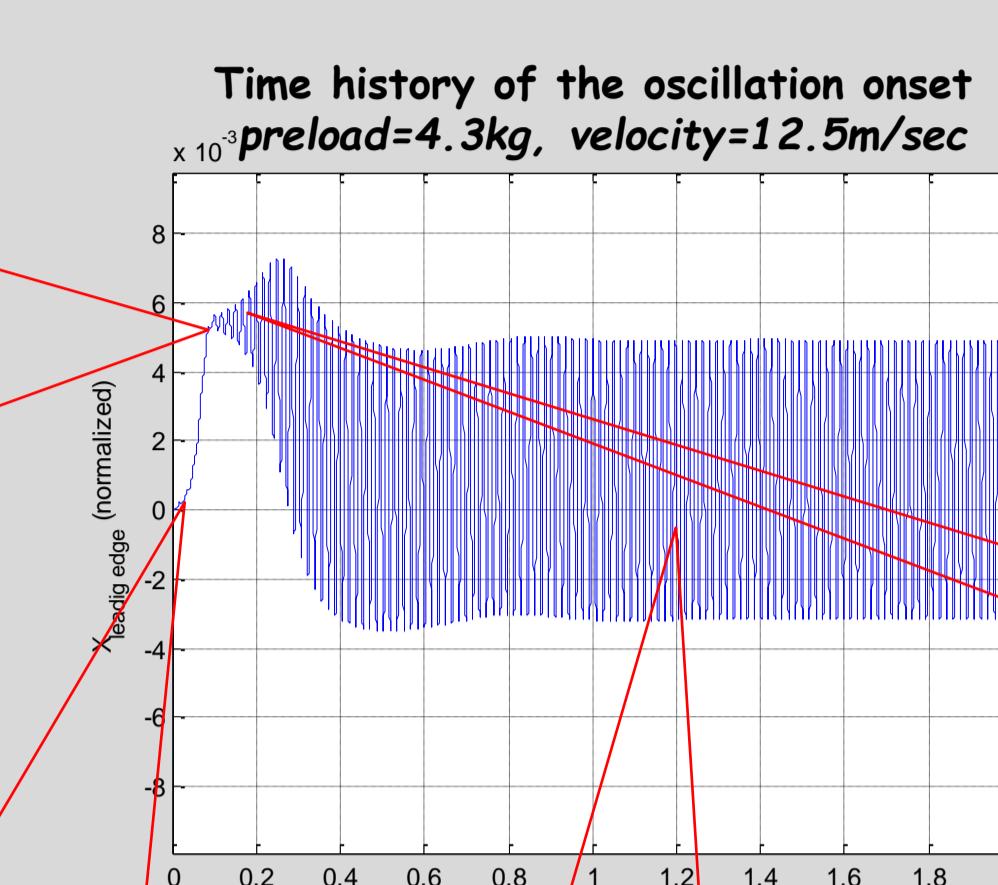
Divergence before flutter (?)

- At high preload values, linear analysis yields divergence before flutter
- Non linear analysis and experiment yield LCO at ~divergence velocity
- Assumed Mechanism:

2. Static stabilization due to nonlinear stiffening

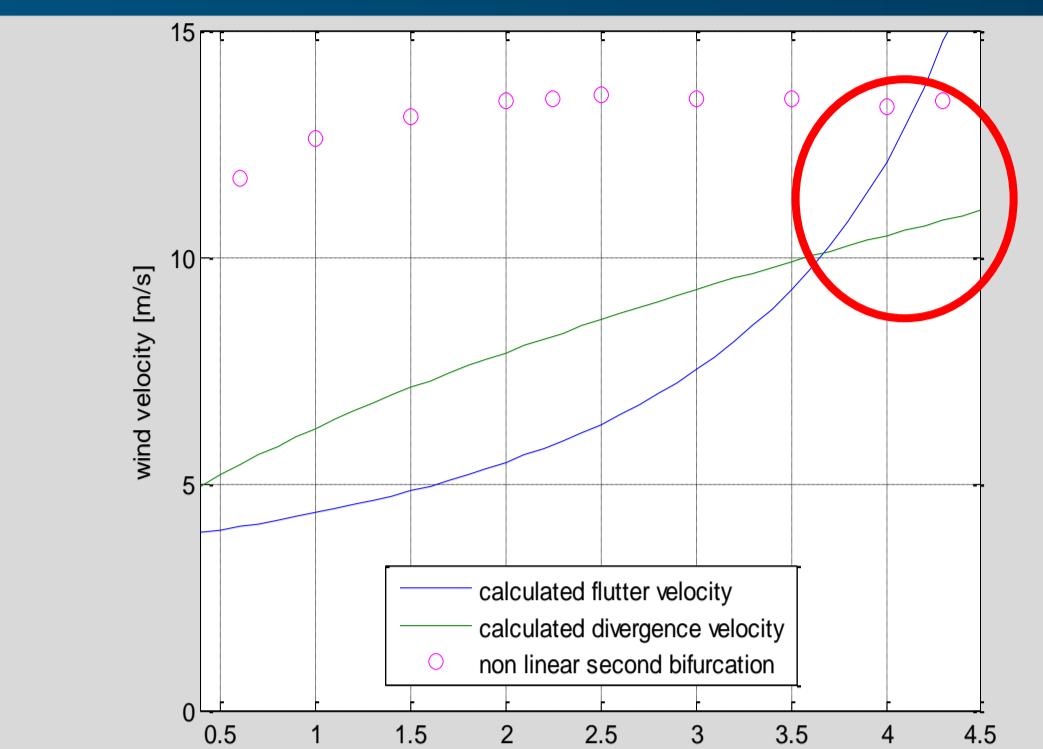
$$F = \frac{1}{2} \rho_s V_0^2 S C_{lx} \alpha$$

$$\alpha = \pm \sqrt{(\rho V_0^2 S e C_{lx} - 2 K_0) / 2 \xi}$$



1. Beginning of divergent movement

4. Flutter converges into LCO



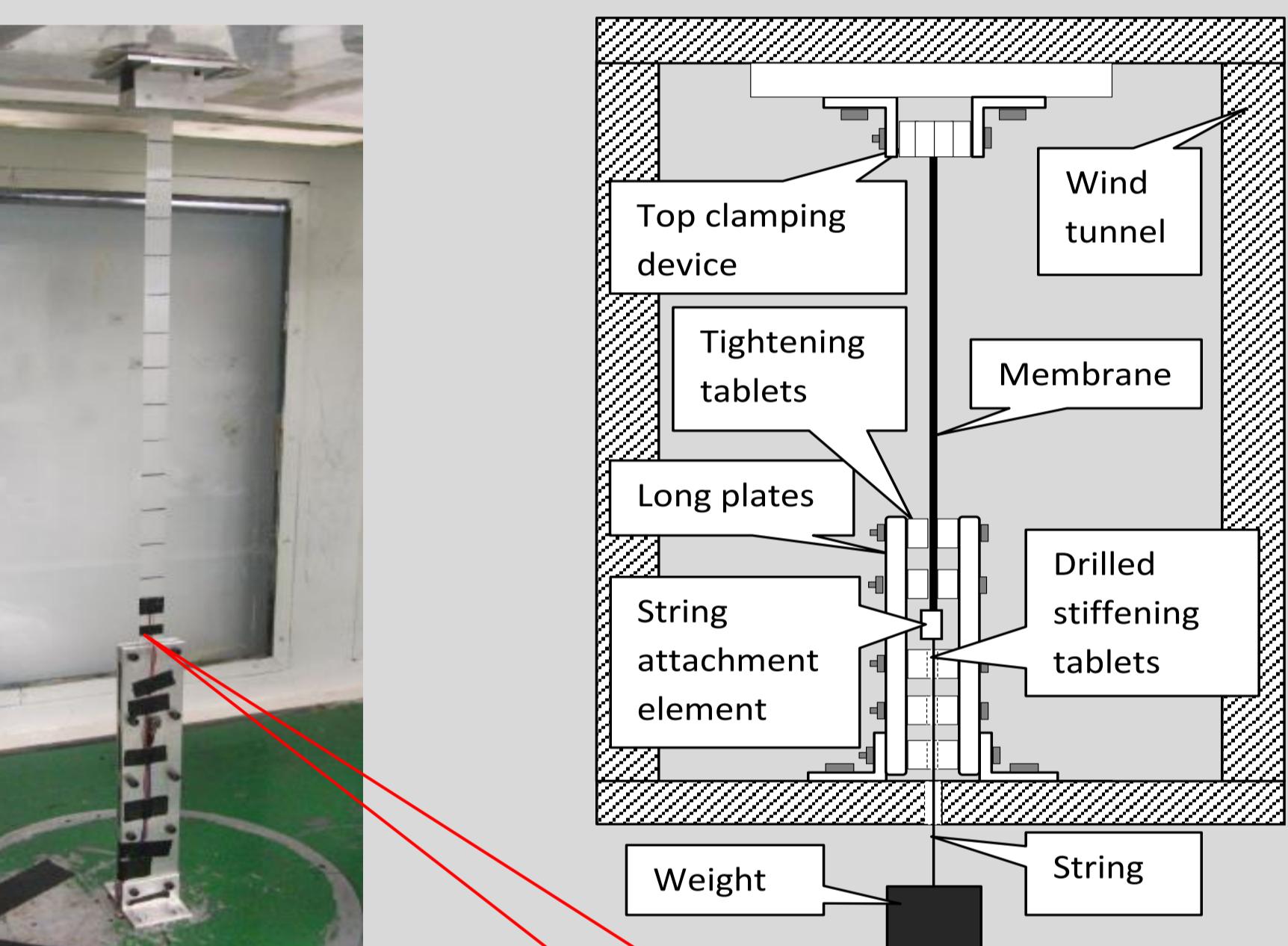
3. Dynamical system changes due to change in stiffness

$$K_{post_divergence} = K_0 + \xi \alpha^2$$

New system I is dynamically unstable leading to flutter

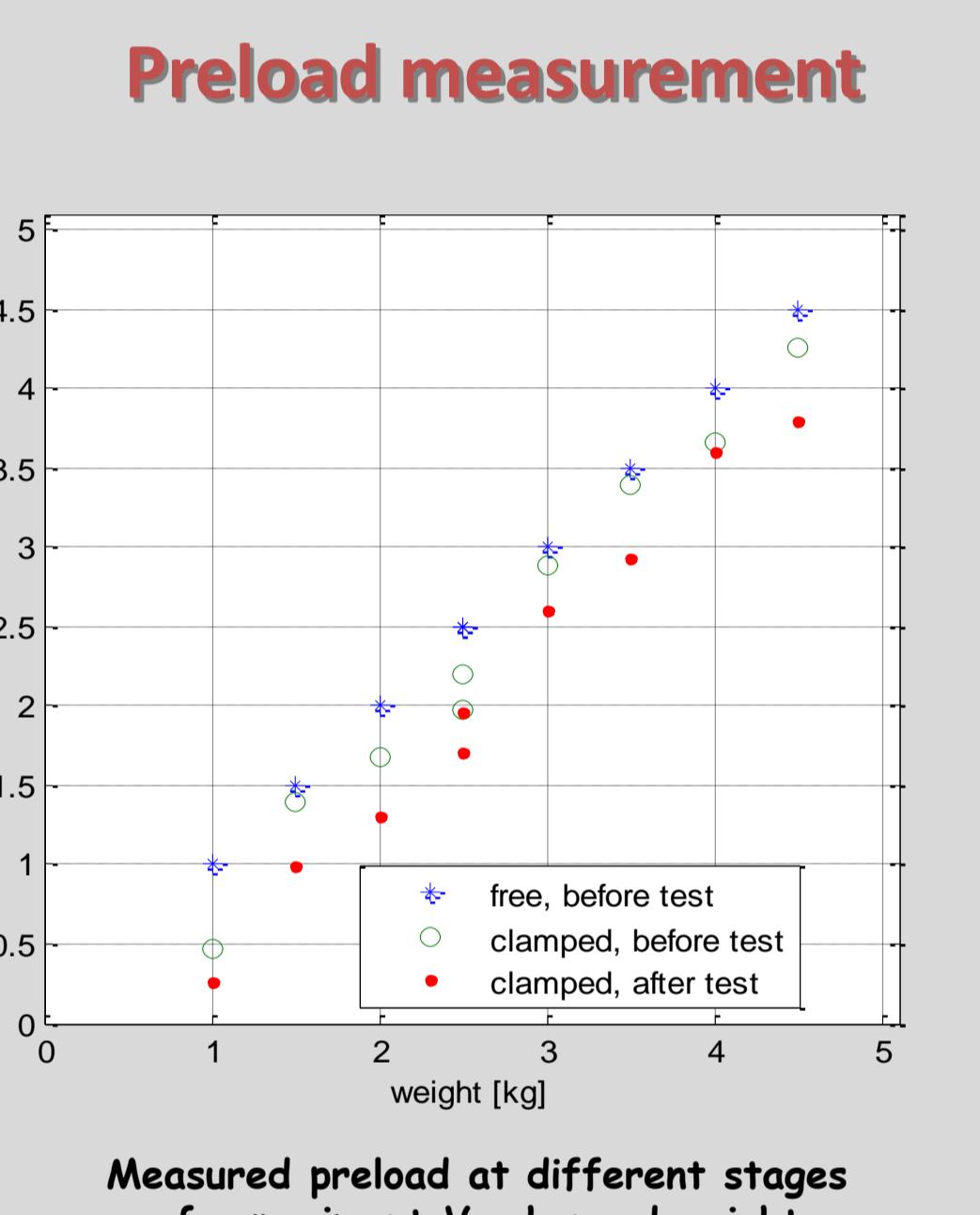
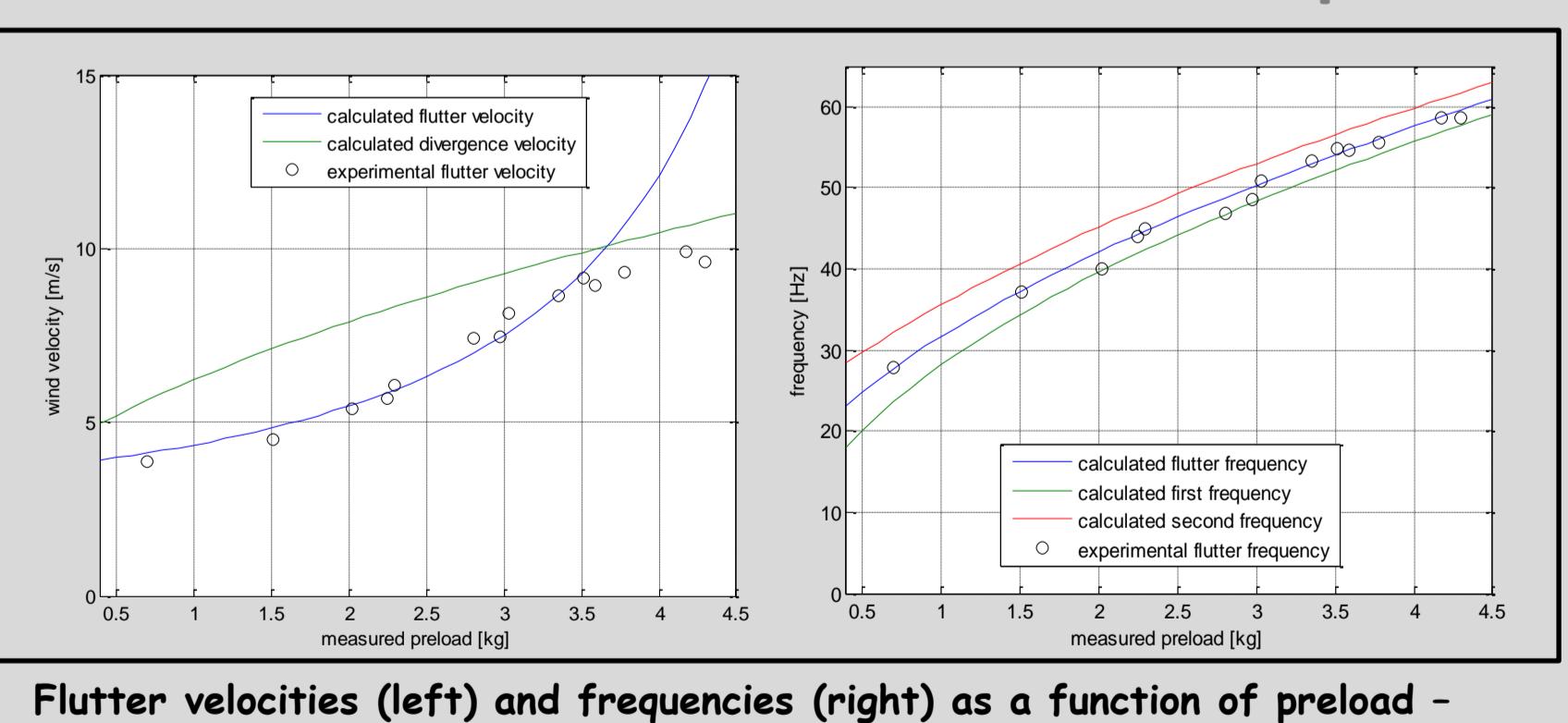
Experimental Study

Test Setup

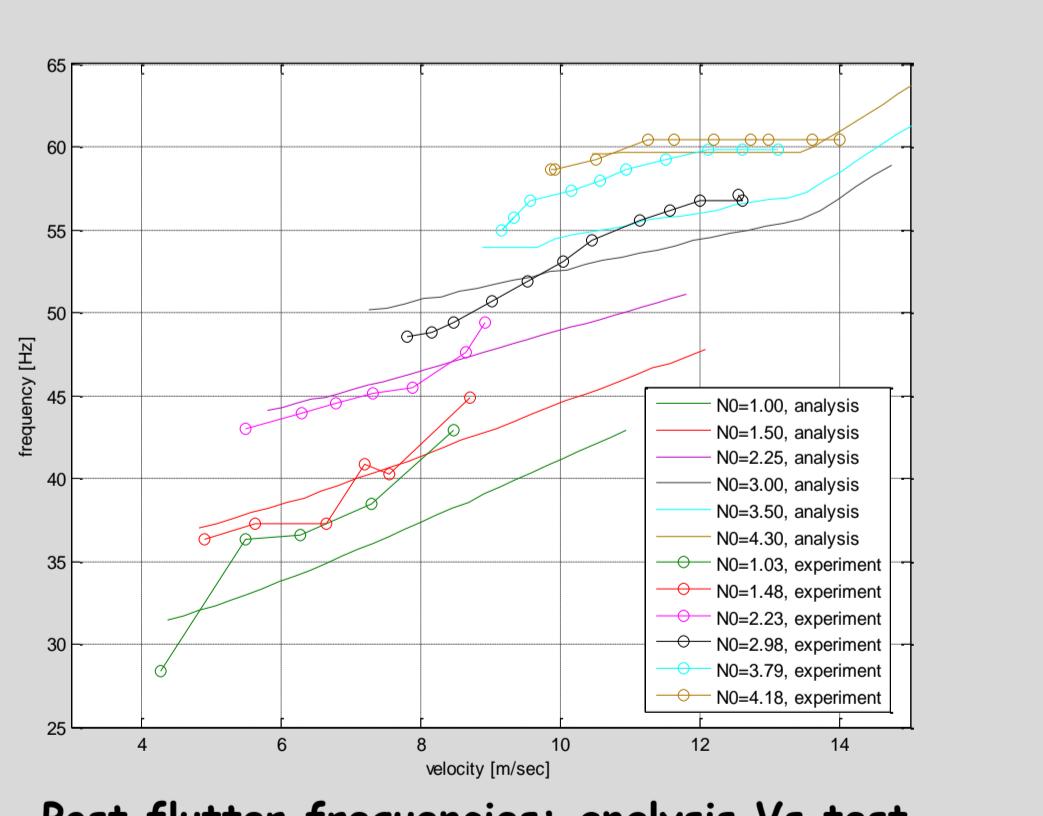


- 0.25/25/600 mm strip
- Weights: 0-5 kg
- Strain gauges ~50 mm from clamping device
- Accelerometer on clamping device
- Acquisition frequency 5 kHz
- Mode shape observed with a stroboscope

LCO onset and frequencies



- Preload is reduced during clamping
- Preload is reduced during Oscillation
- Results are based on the preload after clamping
- Results are more accurate at the beginning of the experiment



Strain amplitude comparison

